



POORNIMA

COLLEGE OF ENGINEERING

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Sample Internal Answer sheet

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A/B. No. PGC/18/.....

Theory



POORNIMA

GROUP OF COLLEGES

Name of Institution..... Poornima College of Engineering.....

Course..... B.Tech.....Branch..... ME.....Year..... 2nd.....Sem..... 3rd.....Sec..... -.....Roll No..... 20/ME/09.....Name of Exam..... 1st mid term.....Day & Date..... 27/Nov/2021, Saturday.....Name of Candidate..... Jagati Agarwal.....College Registration No..... PCE20 ME 007.....Name of Subject/Paper..... Mechanics of solid.....Subject/Paper Code..... 3ME4-01.....Signature of Candidate..... Jagati.....Name & Signature of Invigilator..... [Signature].....

Question No.	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15	Maximum Marks	80	Signature of Examiner
Marks Obtained	2	2	2	2	2	7	7	7			7	14		12	14	Total Marks Obtained	(78)	<u>[Signature]</u>

INSTRUCTIONS :

- No Supplementary answer books will be issued. Write on each ruled line on both sides of the leaf. Please do not waste pages.
- Bringing cell phone / communication devices / programmable calculator (i.e. having memory capacity of more than six numbers) is strictly prohibited. Exam. Conducting authorities will not keep them under their custody.
- During the course of examination the candidate shall be under the discipline & control of the invigilator & shall of invigilators on all matters relating to the examinations.
- Make all due entries on the cover page very carefully only at the space provided for the purpose.

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Seal

part - A

→ Stress is the internal resistance offered by a body to deformation is called stress. (\sim)

$$\text{Stress} = \frac{\text{pressure}}{\text{Area}}$$

Strain is ratio of change in dimension to its original dimension. Denoted by e .

stress

- Tensile stress
- Compressive stress
- ~~→ Volumetric stress~~
- shear stress

strain

- Tensile strain
- Compressive strain
- shear strain
- Volumetric strain

→ $\frac{\text{Temp. strain}}{\Delta L} = \alpha T$

$\frac{\text{Temp. stress}}{\Delta L} = \alpha T E$

↓ E → Modulus of elasticity
coefficient of linear expansion

T = change in Temperature

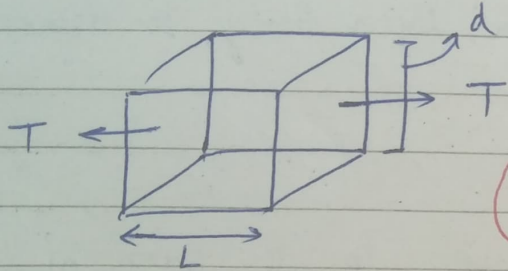
$$\left\{ \begin{array}{l} e \propto T \\ e = \alpha T \end{array} \right\}$$

with an increase in temperature there of body. hence Temperature $\uparrow \Rightarrow$ str

$\therefore e \propto T$

3> $\frac{\text{lateral strain}}{\text{longitudinal strain}} = \frac{1}{m} = \mu$

where μ = poisson's ratio



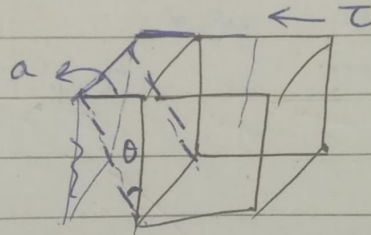
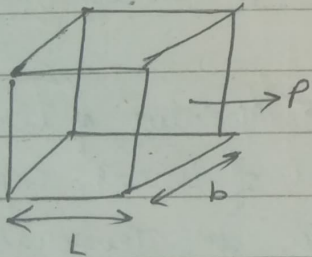
Here,

* longitudinal strain = $\frac{\Delta l}{L} = \frac{\sigma}{E}$

* lateral strain = $-\frac{\mu \sigma}{E} = \frac{\Delta d}{d}$

4> Modulus of Elasticity - Ratio of tensile/compressive stress to tensile/compression strain within elastic limit is (E). Modulus of elasticity.

Modulus of Rigidity - Ratio of shear stress to the shear strain within elastic limit is (G). Modulus of rigidity.



$E = \frac{\sigma}{e} = \frac{P \times L}{A \times \Delta l}$

$G = \frac{\tau}{\theta} = \frac{\text{shear stress}}{\text{shear strain}}$

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$$6 \rightarrow d = 25 \text{ mm}$$

$$P = 40 \text{ kN}$$

$$L = 200 \text{ mm}$$

$$\delta l = 0.085 \text{ mm}$$

$$\delta d = 0.003 \text{ mm}$$

$$E = \frac{\text{stress}}{\text{strain}} = \frac{P}{A(e_1 + e_2)} \quad \text{--- (1)}$$

$$A = \frac{\pi (25)^2}{4} = \underline{\underline{490.873 \text{ mm}^2}}$$

$$e_1 = \frac{\delta l}{l} = \frac{0.085}{200}$$

$$e_2 = \frac{\delta d}{d} = \frac{0.003}{25}$$

$$= \underline{\underline{0.000425 \text{ mm}}}$$

$$= \underline{\underline{0.000120 \text{ mm}}}$$

$$\text{strain} = e_1 + e_2 = 0.000545 \text{ mm}$$

$$\text{Now, Modulus of Elasticity} = \frac{40 \times 10^3}{(490.873) \times 0.000545}$$

$$E = \underline{\underline{149.5182978 \times 10^3 \text{ N/mm}^2}}$$

$$\text{Also, } E = 2G(1 + \mu)$$

$$\rightarrow \mu = \frac{\text{lateral strain}}{\text{longitudinal strain}}$$

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Modulus of rigidity (G) = $\frac{E}{2(1+\mu)}$

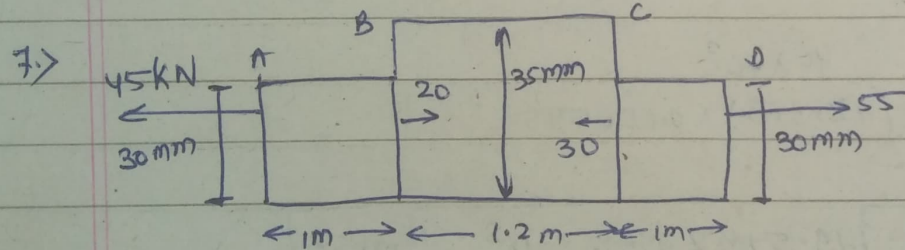
$G = 58,300.825 \text{ N/mm}^2$ (using previous results)

$E = 3K(1-2\mu)$

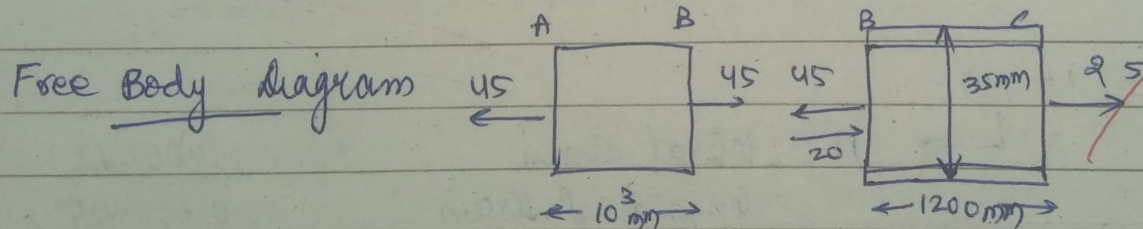
$K = \frac{E}{3(1-2\mu)} = \frac{149.5782978 \times 10^3}{3(1-2(0.2823))}$

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$K = 114,468.1502 \text{ N/mm}^2$
↓
volumetric strain



~~$E = 205 \times 10^3 \text{ N/mm}^2$~~



(Tensile)

(Tensile)

(Tensile)

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$E = 205 \times 10^3$
 $= 205 \times 10^3$
 $= 205 \times 10^3$

$$\begin{aligned} \text{c/s area of AB} &= \frac{\pi}{4} (30)^2 \\ &= 706.858 \text{ mm}^2 \end{aligned}$$

$$\begin{aligned} \text{c/s area of BC} &= \frac{\pi}{4} (35)^2 \\ &= 962.11275 \text{ mm}^2 \end{aligned}$$

$$\frac{\text{Elongation}}{\delta l} = \delta l_1 + \delta l_2 + \delta l_3$$

$$= \frac{1}{E} \left(\frac{P_1 L_1}{A_1} + \frac{P_2 L_2}{A_2} + \frac{P_3 L_3}{A_3} \right)$$

$$A_1 = A_3$$

$$= \frac{1}{205 \times 10^{-3}} \left(\frac{45 \times 10^3 \times 10^3}{706.858} + \frac{25 \times 10^3 \times 1200}{962.11275} + \frac{55 \times 10^3 \times 10^3}{706.858} \right)$$

$$= \frac{10^{5+3}}{205} \left(\frac{450}{706.858} + \frac{25 \times 12}{962.11275} + \frac{550}{706.858} \right)$$

$$= 0.00842207 \times 10^8 \text{ mm}$$

$$\delta l_1 = 310,546.3829; \delta l_2 = 152104.2762;$$

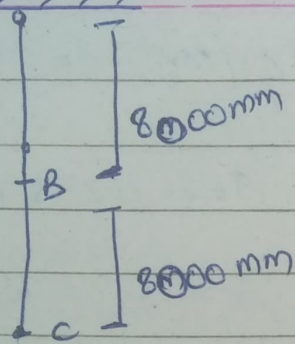
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$$\delta l = 842207.3493 \text{ mm}$$

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$$\begin{aligned} E &= 205 \times 10^3 \text{ N/mm}^2 \\ &= 205 \times 10^3 - 6 \text{ N/mm}^2 \\ &= 205 \times 10^{-3} \text{ N/mm}^2 \end{aligned}$$

6. >



$$A = 4 \text{ mm}^2$$

$$P = 20 \text{ N}$$

$$E = 200 \times 10^3 \text{ N/mm}^2$$

At C, deflection is due to self weight of the material.

$$\frac{\sigma (\text{stress})}{\text{strain}} = E$$

$$P = \text{weight} = 20$$

$$\frac{Pl}{2AE} = \delta l_c$$

$$\frac{20 \times 16000}{2 \times 4 \times 200 \times 10^3} = \delta l_c$$

$$\delta l_c = \frac{1}{5} = 0.2 \text{ mm}$$

Deflection at pt. B is due to self weight produce by AB & load produce by BC

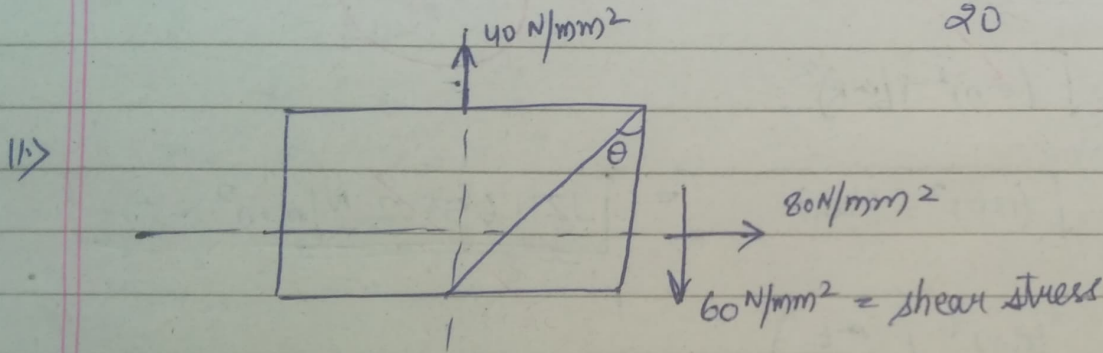
$$\delta l_B = \left(\frac{W/2 \times l/2}{2 \times A \times E} \right)_{AB} + \left(\frac{W/2 \times l/2}{A \times E} \right)_{BC}$$

$$\delta l_B = \frac{w/2 \times l/2}{AE} \left(\frac{1}{2} + 1 \right) \Rightarrow \frac{wl}{4AE} \times \frac{3}{2} \Rightarrow \frac{3wl}{8AE}$$

(7)

$$= \frac{3 \times 20 \times 16 \times 10^3}{8 \times 4 \times 200 \times 10^3}$$

$$= \frac{3}{20} = 0.15 \text{ mm} = 8 \text{ lb}$$



$$\theta = 45^\circ$$

$$\sigma_1 = 80 \text{ N/mm}^2$$

$$\sigma_2 = 40 \text{ N/mm}^2$$

$$\tau = 60 \text{ N/mm}^2$$

Find - $\sigma_n, \sigma_t, \sigma_R, \tan \phi = \frac{\sigma_t}{\sigma_n}$ or $\cot \phi = \frac{\sigma_n}{\sigma_R}$

normal stress (σ_n)

using formula = $\frac{\sigma_1 + \sigma_2}{2} + \frac{\sigma_1 - \sigma_2}{2} \cos 2\theta$

$$= \frac{80 + 40}{2} + \frac{(80 - 40)}{2} \cos 2\theta$$

$$\sigma_n = 120 \text{ N/mm}^2$$

$\cos 2(45^\circ)$
 $\cos 90^\circ = 0$

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shear stress (σ_t) = using formula

$$= \frac{\sigma_1 - \sigma_2}{2} \sin 2\theta = \tau \cos 2\theta$$

$$= \frac{80 - 40}{2} \sin(90)$$

$$\sigma_t = 20 \text{ N/mm}^2$$

$$\text{Resultant } (\sigma_R) = \sqrt{(\sigma_n)^2 + (\sigma_t)^2}$$

$$= \sqrt{(120)^2 + (20)^2} = 121.655 \text{ N/mm}^2 = \sigma_R$$

$$\text{Obliquity } (\phi) = \tan^{-1} \left(\frac{\sigma_t}{\sigma_n} \right)$$

$$= \tan^{-1} \left(\frac{20}{120} \right)$$

$$\phi = 9.4623^\circ$$

$$\phi \approx 9.5^\circ$$

Angle b/w σ_n & σ_R

$$\phi = 9.4623$$

$$\left\{ \cos^{-1} \left(\frac{\sigma_n}{\sigma_R} \right) \right\}$$

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part - c

$$12. \rightarrow A_r = \frac{\pi}{4} (20)^2 = 314.159 \text{ mm}^2$$

$$E = 2 \times 10^5 \text{ N/mm}^2$$

$$A_t = \frac{\pi}{4} (30^2 - 25^2) = 215.9844 \text{ mm}^2$$

$$L = 800 \text{ mm}$$

$$P = 20 \times 10^3 \text{ N}$$

$$\sigma_t \text{ \& \& } \sigma_r = ?$$

$$\sigma_{t_{\text{new}}} \text{ \& \& } \sigma_{r_{\text{new}}} = ?$$

$$\left[\delta l = \frac{10}{4} \text{ mm} \right] \text{ given}$$

A_r = area of rod

A_t = area of tube

σ_r = stress in rod

σ_t = stress in tube

As per given condition, at static equilibrium load are equal

$$\sigma_r A_r = \sigma_t A_t$$

$$\sigma_r (314.159) = \sigma_t \times 215.9844$$


$$\sigma_r = \left(\frac{900 - 625}{400} \right) \sigma_t$$

$$\sigma_r = 0.6875 \sigma_t \quad \text{--- (1)}$$

Also, given load on tube = $20 \times 10^3 \text{ N}$

$$P_t = \sigma_t A_t = 20 \times 10^3$$

$$\Rightarrow \sigma_t = \frac{20 \times 10^3}{215.9844}$$


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$$\sigma_t = \frac{4 \times (20 \times 10^3)}{\pi \times (275)} = 92.599 \text{ N/mm}^2$$

By 1st eqⁿ

$$\sigma_s = 0.6875 (\sigma_t)$$

$$\sigma_s = 63.661977 \text{ N/mm}^2$$

Now, new stress of new nuts & bolts is calculated by given condition

$$S_l = \frac{10}{4} \text{ mm}$$

6 nut is tightened by 1 Quarter hence, $S_l = \frac{10}{4} \left(\frac{1}{4} \right)$

$$\frac{\sigma_s L}{E} + \frac{\sigma_t L}{E} = \frac{10}{4} \left(\frac{1}{4} \right) \quad \left(\sigma_s = 0.6875 \sigma_t \right)$$

$$L = 800 \text{ mm}$$

$$E = 2 \times 10^5 \text{ N/mm}^2$$

$$(1.6875) \sigma_t = \frac{10}{4} \left(\frac{1}{4} \right) \times \frac{E}{L}$$

$$\sigma_t = \frac{10}{16} \times \frac{2 \times 10^5}{800} \times \frac{1}{1.6875}$$

$$\sigma_t = \frac{0.0370370 \times 10^6}{4 \times 10^2} = 92.59259 \text{ N/mm}^2$$

$$\sigma_s^* = \underline{63.6574 \text{ N/mm}^2}$$

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$$14. \rightarrow A_s = \frac{\pi}{4} \left((30)^2 - (20)^2 \right) = \underline{125 \pi \text{ mm}^2}$$

$$A_c = \frac{\pi}{4} (15)^2 = \underline{56.25 \pi \text{ mm}^2}$$

$$\Delta T = 190^\circ \text{C}$$

$$E_s = 2.1 \times 10^5 \text{ N/mm}^2$$

$$E_c = 1 \times 10^5 \text{ N/mm}^2$$

$$\alpha_s = 11 \times 10^{-6} \text{ per } ^\circ \text{C}$$

$$\alpha_c = 18 \times 10^{-6} \text{ per } ^\circ \text{C}$$

$\alpha_c > \alpha_s$
 \downarrow compress \downarrow Tensile

At static equilibrium, load is equal.

$$\sigma_s A_s = \sigma_c A_c$$

$$\sigma_s (125 \pi) = \sigma_c (56.25) \pi$$

$$\boxed{2.22 \sigma_s = \sigma_c}$$

By Thermal expansion method

$$\frac{\sigma_s}{E_s} \left(\alpha_s \Delta T + \frac{\sigma_s}{E_s} \right) \frac{\pi}{4} = \left(\alpha_c \Delta T + \frac{\sigma_c}{E_c} \right) \frac{\pi}{4}$$

\downarrow (tve) (Tensile) \downarrow (-ve) \rightarrow as compo

$$= (11 \times 10^{-6} \times 190) + \frac{\sigma_s}{2.1 \times 10^5} = + \frac{(2.22 \sigma_s) \times 190}{10^5} - (18 \times 10^{-6} \times 190)$$

$$5510 \times 10^{-6} = \frac{\sigma_s}{10^5} \left(\frac{1}{2.1} + 2.22 \right)$$

$$\sigma_s = \frac{5510 \times 10^{-1} \times 2.1}{5.662}$$

$$= \boxed{204.362416 \text{ N/mm}^2 = \sigma_s}$$

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$$\sigma_E = 2.22 \sigma_s$$

$$\sigma_C = \boxed{453.68 \text{ N/mm}^2}$$

15. > Make axis

2. > Mark pt. A on it

3. > Measure AC = a_1 & AB = a_2 on the axis

4. > From B & C measure 2.5 cm. ~~from~~ \perp^x to show shear stress

5. > Taking EO as radius make wicde

6. > Measure 20 angle as shown in figure

7. > Which gives pt. E

8. > Make \perp^x from E = σ_t & make mark it as σ_t

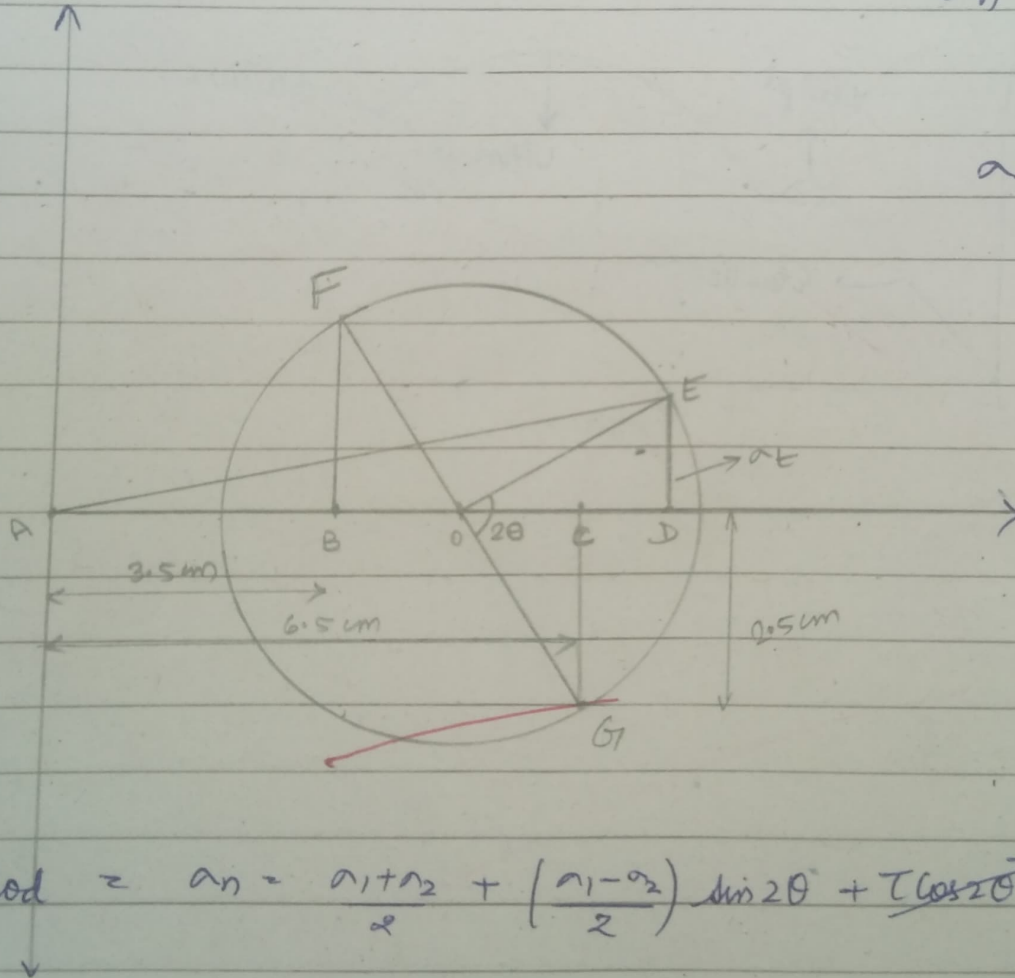
9. > Measure AD = normal stress

$\sigma_1 = 65 \text{ N/mm}^2 = 6.5 \text{ cm}$ (Tensile) Scale $10 \text{ N/mm}^2 = 1 \text{ cm}$
 $\sigma_2 = 35 \text{ N/mm}^2 = 3.5 \text{ cm}$ (Tensile)
 $\tau = 25 \text{ N/mm}^2 = 2.5 \text{ cm}$
 $\theta = 45^\circ$

$\sigma_n = AD = 7.5 \text{ cm}$

$\sigma_n = 75 \text{ N/mm}^2$

$\sigma_t = ED = 1.5 \text{ cm}$
 $= 15 \text{ N/mm}^2$



14

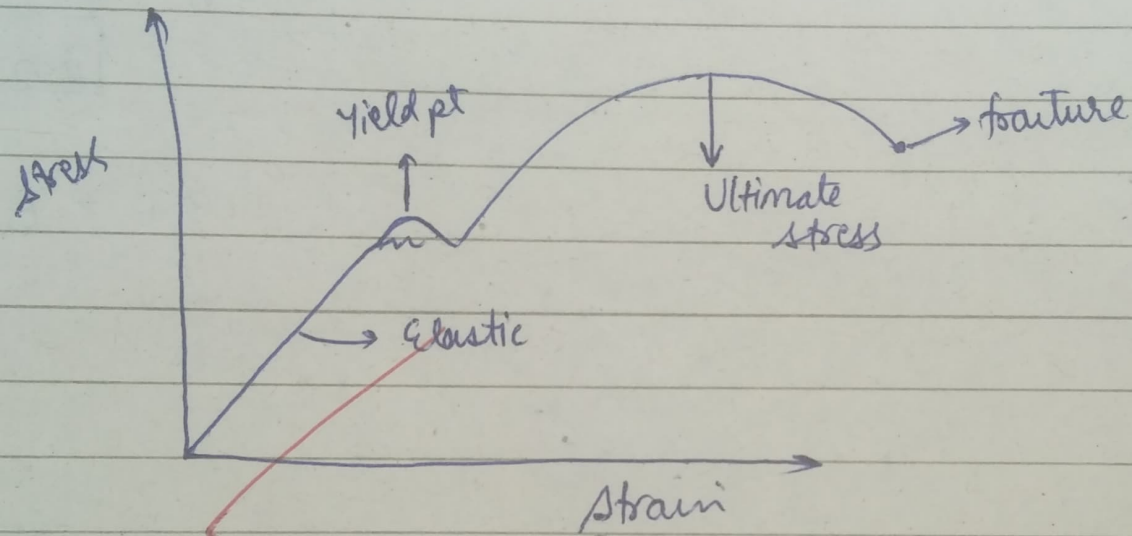
Theoretical method $\sigma_n = \frac{\sigma_1 + \sigma_2}{2} + \left(\frac{\sigma_1 - \sigma_2}{2} \right) \cos 2\theta + \tau \sin 2\theta$
 $= 75 \text{ N/mm}^2$

$\sigma_t = \left(\frac{\sigma_1 - \sigma_2}{2} \right) \sin 2\theta - \tau \cos 2\theta = 15 \text{ N/mm}^2$

part-A

mild steel shows ductile nature
↓

undergoes plastic deformation by Tensile load without fracturing -



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